

## References

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## Approximate Expression for the Boundary-Layer Shape Factor

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## Introduction

THE momentum integral equation of boundary-layer analysis contains a shape factor, which is the ratio of displacement thickness to momentum thickness. With the aid of approximate velocity and temperature profiles, the shape factor may be obtained as a function of the external stream quantities and the profile parameters. Unfortunately, the resulting function generally is not analytical. This note considers the shape factor for the case where the velocity profile is given by a power law and the temperature profile by the Crocco relation; an accurate but simple analytical approximation to the shape factor is described.

## Evaluation and Approximation of the Shape Factor

The boundary-layer shape factor  $H$  is defined by

$$H \equiv \frac{\delta^*}{\theta} \equiv \frac{\int_0^1 \left(1 - \frac{\rho u}{\rho_e u_e}\right) d\left(\frac{y}{\delta}\right)}{\int_0^1 \left(1 - \frac{u}{u_e}\right) \frac{\rho u}{\rho_e u_e} d\left(\frac{y}{\delta}\right)} \quad (1)$$

where the subscript  $e$  denotes quantities at the external edge of the boundary layer.

In both laminar and turbulent compressible flows the velocity profile can be approximated by

$$u/u_e = (y/\delta)^{1/N} \equiv z \quad (2)$$

When the Prandtl number of the fluid is unity and there is either no pressure gradient or no heat transfer from the wall, the Crocco relation holds. For a perfect gas this relation may be written

$$T^o - T_w = (u/u_e) - (T_e^o - T_w) = z(T_e^o - T_w) \quad (3)$$

where  $T_w$  is the temperature at the wall and superscript  $o$  denotes stagnation values. Walz<sup>1</sup> suggests that Eq. (3) is also valid for flow over a nonadiabatic wall where the pressure gradient is moderate. From the definitions of Mach number and stagnation enthalpy it follows that

$$\frac{T}{T_e} = f(z; m, t) \equiv \frac{1+m}{t} + \frac{t-1}{t} (1+m)z - mz^2 \quad (4)$$

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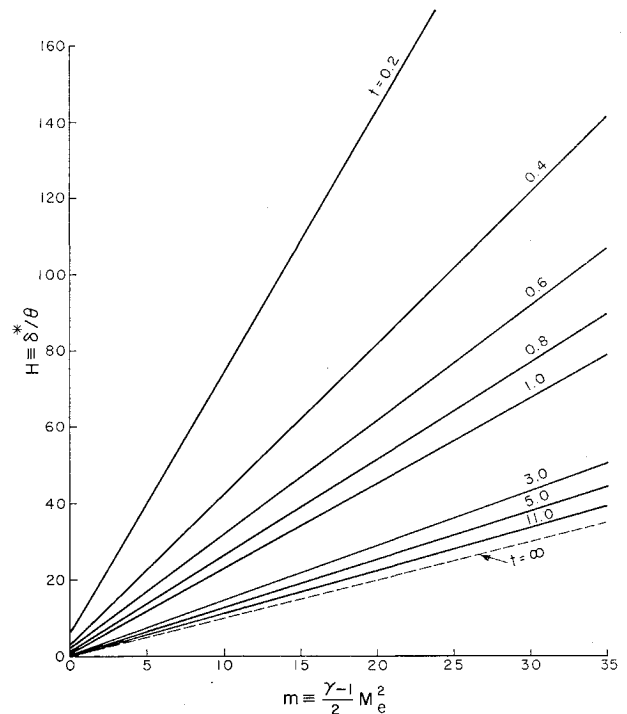


Fig. 1 The variation of shape factor with flow conditions when the velocity profile obeys a  $\frac{1}{N}$  power law.

where  $t \equiv T_e^o/T_w$  and  $m \equiv [(\gamma - 1)/2]M_e^2$ . Using Eqs. (2) and (4) and the perfect gas law, Eq. (1) may be rewritten in the form

$$H = H(m; t, N) = \frac{\int_0^1 \left[1 - \frac{z}{f(z; m, t)}\right] z^{N-1} dz}{\int_0^1 \left[\frac{1-z}{f(z; m, t)}\right] z^N dz} \quad (5)$$

Bartz<sup>2</sup> has evaluated numerically the integrals in Eq. (5) for the case  $N = 7$ . Using Bartz's values, the function  $H(m; t, 7)$  has been calculated and is shown in Fig. 1. As indicated by Fig. 1, the function is approximately linear in  $m$  for fixed values of  $t$ . The error in a linear representation never exceeds 2% and is less than 0.5% for  $m > 16$ . The integrals in Eq. (5) have been evaluated analytically for the case of  $N = 1$  (Ref. 3), and the values of  $H$  again lie almost exactly on a family of straight lines. Tucker<sup>4</sup> has evaluated  $H$  for  $t = 1$  and a number of values of  $N$  in the range 5 to 11. The values of  $H(m; 1, N)$  again turn out to be approximately linear in  $m$ . Finally, the limit  $N \rightarrow \infty$  of Eq. (5) has been taken (see Ref. 3 for details), leading to the result

$$H(m; t, \infty) = 1/t + (1 + 1/t)m \quad (6)$$

which is exactly linear in  $m$  for fixed  $t$ .

It may therefore be concluded that a good approximation for  $H$  is

$$H(m; t, N) = H_0(t, N) + H_1(t, N)m \quad (7)$$

Fitted values of  $H_0$  and  $H_1$  for  $n = 1$  and 7 and analytical values of  $H_0$  and  $H_1$  for  $N = \infty$  are shown in Figs. 2 and 3 together with values for  $N = 5$  and  $N = 11$  at  $t = 1$ .

The shapes of the curves in Figs. 2 and 3 suggest that  $H_0$  and  $H_1$  may be asymptotic to 0 and 1, respectively, for large  $t$ . This can be shown to be true by rearranging Eq. (1) into the form

$$H = \frac{\int_0^1 \left\{ \left[ \frac{(1+m)/t + (mt-1-m)z/t - mz^2}{f(z; m, t)} \right] z^{N-1} \right\} dz}{\int_0^1 \left[ \frac{(1-z)}{f(z; m, t)} \right] z^N dz} \quad (8)$$

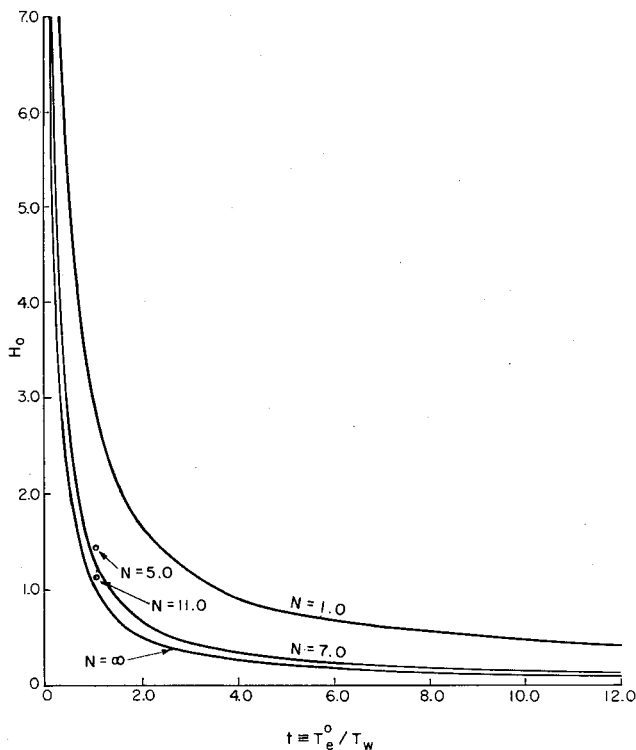


Fig. 2 The variation of  $H_0$  with temperature ratio for different velocity profiles.

Taking the limit as  $t \rightarrow \infty$  gives

$$H(m; \infty, N) = m \quad (9)$$

and this limiting equation for  $H$  has been included in Fig. 1. For  $t \ll 1$

$$f(z) \approx (1 + m)(1 - z)/t \quad (10)$$

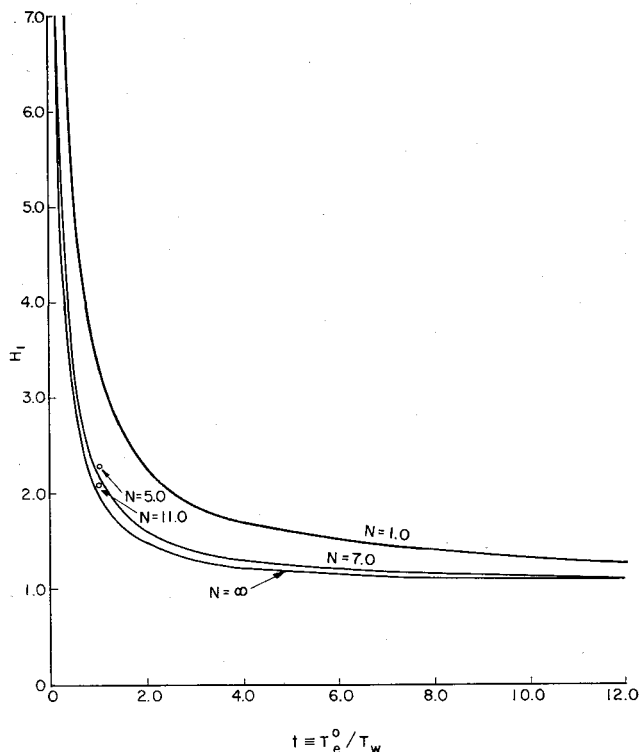


Fig. 3 The variation of  $H_1$  with temperature ratio for different velocity profiles.

and Eq. (5) may be integrated to give

$$H \approx [(N + 1)/Nt](1 + m) \quad (11)$$

which also exhibits the suggested linearity in  $m$ .

Thus Figs. 2 and 3 and Eq. (9) show that at moderately large values of  $t$  (as found in hypersonic flow),  $H$  is almost independent of  $t$ . In turbulent flow,  $N$  generally lies in the range 5 to 11 and for compressible laminar flow,  $N = 1$  is a reasonable approximation. Figures 2 and 3 show that, except at low values of  $t$ , change of  $N$  has only a small effect on  $H_0$  and  $H_1$  with laminar flow (low  $N$ ) being more sensitive than turbulent flow (high  $N$ ).

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## Experiments on a Faraday-Type MHD Accelerator with Series-Connected Electrodes

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## Introduction

SINCE a conventional segmented-electrode magnetohydrodynamic (MHD) accelerator or generator develops an axial Hall electric field, the potential on a given anode may be equal to the potential on some preceding cathode in an accelerator or some following cathode in a generator. With this in mind, de Montardy<sup>1</sup> has suggested connecting the electrodes of a generator in series as illustrated in Fig. 1; thereby making it a two-terminal power supply. In order to simplify the external connections, Dicks<sup>2</sup> has extended this idea by pointing out that the sidewalls in MHD generators or accelerators might be made up of metallic strips that lie approximately along the equipotential lines and connect anodes and cathodes that have the same potential. These diagonal metal strips in the walls would replace the jumpers shown in Fig.

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